



10AU65

Sixth Semester B.E. Degree Examination, Jan./Feb. 2021
Heat and Mass Transfer

Time: 3 hrs.

Max. Marks:100

- Note:1. Answer any FIVE full questions, selecting atleast TWO questions from each part.
2. Heat transfer data book and steam tables are permitted

PART – A

- 1 a. What do you mean by boundry condition of 1st, 2nd and 3rd kind? (06 Marks)
b. Derive the general three dimensional heat conduction equation in Cartesian co-ordinate system. (08 Marks)
c. A wall of a furnace is made up inside layer of silica brick 120mm thick covered with a layer of magnesite brick 240mm thick. The temperature at the inside surface of silica brick wall and outside surface of magnesite brick walls are 725^oC and 110^oC respectively. The contact thermal resistance between two walls at the interface is 0.0035^oC/w per unit wall area. If the thermal conductivities of silica and magnesite bricks are 1.7w/m^oC and 5.8 w/m^oC. Calculate i) The rate of heat loss per unit area of walls.
ii) The temperature drop at the interface. (06 Marks)
- 2 a. Clearly define : i) Fin efficiency ii) Fin effectiveness. (04 Marks)
b. Derive an expression for temperature distribution for rectangular fin, with the tip insulated. (08 Marks)
c. A wire of 6.5mm diameter at a temperature of 60^oC is to be insulated by a material having K = 0.174 w/m^oC. Convection heat transfer coefficient h_o = 8.722 w/m² ^oC. The ambient temperature is 20^oC. For maximum heat loss what is the maximum thickness of insulation and heat loss per meter length? Also find the percentage increase in the heat dissipation. (08 Marks)
- 3 a. Define : i) Biot number and ii) Fourier number. (04 Marks)
b. Show that the temperature distribution under lumped system analysis is given by
$$\frac{T - T_{\infty}}{T_1 - T_{\infty}} = e^{-BiFo}$$
 (08 Marks)
c. A thermocouple the function of which can be approximated as a 1mm diameter sphere is to measure the temperature of a gas stream. The properties of the function are $\rho = 8500 \text{ kg/m}^3$, $C = 320 \text{ J/kg K}$ and $K = 35 \text{ w/m K}$. The heat transfer co-efficient between the junction and the gas is 210 w/m² K. Determine how long it will take for the thermocouple to read 99% of the initial temperature difference. (08 Marks)
- 4 a. Briefly explain :
i) Hydrodynamic boundary layer ii) Thermal boundary layer. (04 Marks)
b. Define i) Nusslet number ii) Prandtl number iii) Grashoff Number. (06 Marks)
c. A 15cm outer diameter steel pipe lies 2m vertically and 8m horizontally in a large room with an ambient temperature of 30^oC. If the pipe surface is at 250^oC and the emissivity of steel is 0.6. Calculate the total rate of heat loss from the pipe to the atmosphere. Properties of air at 140^oC are $\rho = 0.854 \text{ kg/m}^3$, $C_p = 1.01 \text{ kJ/kg K}$, $K = 0.035 \text{ w/m K}$, $P_r = 0.684$ and $V = 27.8 \times 10^{-6} \text{ m}^2/\text{sec}$. (10 Marks)

PART – B

- 5 a. Air at 20°C and atm flows over a flat plate at 40m/s . The Plate is 80cm long and is maintained at 60°C . Assuming unit depth in Z – direction, calculate the heat transfer rate from the plate. Properties of air at 40°C are : $P_r = 0.7$, $K = 0.02723 \text{ w/m K}$, $C_p = 1.007 \text{ kJ/kg K}$ and $\mu = 1.906 \times 10^{-5} \text{ kg/ms}$. (10 Marks)
- b. The crank case of an IC engine measuring $0.8\text{m} \times 0.2\text{m}$ may be assumed as a flat plate. The engine runs at a speed at 25m/sec and the crank case is cooled by the air flowing past it at the same speed. Calculate the heat loss from the crank surface maintained at 85°C to the ambient air at 15°C . Due to road induced vibration, the boundary layer becomes turbulent from the leading edge itself. (10 Marks)
- 6 a. Derive an expression for LMTD of a counter flow heat exchanger. (10 Marks)
- b. The flow rates of hot and cold water streams running through a parallel flow heat exchanger are 0.2kg/sec and 0.5kg/sec respectively. The inlet temperature on the hot and cold sides are 75°C and 20°C respectively. The exit temperature of hot water is 45°C . if the individual heat transfer coefficient on both sides are $650 \text{ w/m}^2 \text{ }^{\circ}\text{C}$. Calculate the area of the heat exchanger. (10 Marks)
- 7 a. With the help of boiling curve, explain the different regimes of boiling. (08 Marks)
- b. With neat sketch, explain i) Film condensation ii) Drop wise condensation. (06 Marks)
- c. Water at atmosphere pressure is to be boiled in a polished copper pan, the diameter of the pan is 350mm and is kept at 115°C . Calculate the following : i) Power of the burner ii) Rate of evaporation in kg/sec iii) Critical heat flux for these condition. (06 Marks)
- 8 a. With usual notation, prove that the emissive power of a diffuse surface is equal to π times intensity of radiation. (08 Marks)
- b. Clearly define : i) Black body ii) Planck's law iii) Weins displacement. (06 Marks)
- c. Determine the radiant heat exchange in w/m^2 between two large parallel steel plates of emissive power 0.8 and 0.5 held at a temperature of 1000 K and 500 K respectively. If then copper plate of emissivity 0.1 is introduced as a radiation shield between the two plates. (06 Marks)
